# Sensor fault detection and isolation for electro-mechanical actuators in a reusable launch vehicle TVC system

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# Abstract

This paper introduces a model-based Fault Detection and Isolation (FDI) approach for a Reusable Launch Vehicle (RLV) Thrust Vector Control (TVC) system operated by Electro-Mechanical Actuators (EMAs). The focus is on the sensors required for the EMA embedded control system to track the on-board computer control commands. The nullspace FDI method is considered and applied to detect and isolate additive faults affecting the mentioned sensors. A detailed formulation of the problem and the EMA-based TVC system modelling for FDI synthesis is provided, including the mechanical load exerted by the rocket nozzle. The FDI synthesis framework is introduced and the application of the nullspace-based strategy is described, including considerations about isolability of the faults. Vehicle-induced loads can potentially disrupt the fault detection process, therefore they are included in the problem formulation to achieve decoupling from the residual generator output and not incur into false alarms. The generator performance is then assessed in fault-free and faulty scenarios using a high-fidelity TVC physical model, and successively benchmarked at the example of an RLV mission scenario.

# 1. Introduction

The R&D for Vertical Take-off Vertical Landing (VTVL) Reusable Launch Vehicles (RLV) is gaining a high momentum worldwide. In Europe, several technology demonstrators and commercial launchers are being developed, like CAL-LISTO,<sup>1,2</sup> Themis,<sup>3</sup> or RETALT;<sup>4</sup> on the other hand, the Ariane Next reusable launcher development program aims at achieving reusability in a commercial context. To bring RLV reliability to sufficient standards, suitable algorithms for fault handling and recovery from upset conditions must be investigated. As such, performing Fault Detection and Isolation (FDI) mechanisms are essential to let the onboard software gather awareness of an in-mission performance degradation. The reason why fault detection, diagnosis and recovery are relevant matters for these vehicles is that all mission phases pose harsh challenges to the vehicle's G&C since the performance requirements are usually very strict; as a consequence, any malfunction in a sensor or actuator, or an unforeseen vehicle structural change, could be disruptive, leading to mission degradation or to the loss of vehicle, even with a control system designed for the maximum achievable robustness. It is important therefore to provide software logic able to detect and diagnose a fault, which in turn can lead to an advantage if this additional knowledge is appropriately exploited. The engine's Thrust Vector Control system (TVC) is a critical vehicle control mean; to drive it, Electro-Hydraulic Actuators (EHAs)<sup>5</sup> or Electro-Mechanical Actuators (EMAs)<sup>6</sup> can be employed. The paper focuses on TVC systems based on EMAs. Electrical power drives are progressively tending to replace fluid power drives, especially in aeronautics where the goal is to achieve sufficient reliability for EMAs to completely remove the hydraulic actuators also for the primary control surfaces.<sup>7-9</sup> Launch vehicles can benefit of such an implementation because a large fraction of mass, space, and cost can be saved with respect to the more conventional hydraulic actuators.<sup>5,10</sup> Their applicability has been proven, for example, by the expendable European VEGA rocket<sup>11–13</sup> over two decades of operational life, or by the Electron rocket.6

Several measurements are necessary in EMAs for correct operation, and to comply with damping and response speed requirements. An LVDT<sup>1</sup> stroke displacement sensor, an electric motor encoder and a current sensor are normally

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<sup>&</sup>lt;sup>1</sup>Linear Variable Differential Transformer.

included; faults affecting these devices must be detected and counteracted, in order to accommodate the fault or switch to a redundant physical or virtual sensor for the purpose of not deteriorating the EMA closed-loop performance and preventing bad tracking of the nozzle deflection commands. This is also justified by the fact that EMAs cannot be easily switched to passive mode whenever they experience a physical fault, therefore it is reasonable to start the investigation from the sensor units, whose degradation can be very detrimental while the EMA is still mechanically fully operative. Several strategies to detect faults can be employed. Model-based FDI methods can be considered as more effective than signal-based or data-based techniques because they enable more types of faults to be addressed at the same time. They work by generating a residual signal as difference between measurements and the predictions of adequate mathematical models. These residuals can then be used to determine if a fault has occurred when compared to suitable thresholds.

#### 1.1 State-of-the-art and objectives

Several model-based FDI techniques have been extensively studied over the last decades and applied to numerous problems. In classical hydraulic flight actuators, advanced methods have been proposed by Ossmann and Varga *et al.* using LPV techniques.<sup>14–16</sup> Ossmann *et al.* also introduce a sensor monitoring technique to solve the angle of attack sensor fault isolation problem for optimal aircraft handling.<sup>14</sup> Specifically for electro-mechanical actuators, Balaban *et al.* have proposed a diagnostic algorithm based on an artificial neural network to diagnose EMA sensor faults and classify component failures.<sup>17</sup> Della Vedova *et al.* proposed a model-based FDI method for the prognostics of EMAs using genetic algorithms.<sup>18</sup> Ossmann and van der Linden introduce a model-based FDI approach using linear synthesis techniques to detect EMA sensor faults.<sup>19</sup>

In this work, one of the most consolidated strategies for FDI is considered and applied to an EMA-based TVC system. It employs the nullspace approach proposed by Varga<sup>14,20–22</sup> to detect and isolate additive faults affecting the mentioned sensors. The main advantage of the method is that, by using linear synthesis models in LTI (or LPV) form,<sup>16,19,23</sup> it enables the direct decoupling of both disturbances and inputs from the residuals, and provides linear residual generators (or filters) of minimal order, reducing the implementation costs on the final hardware; additionally, uncertainties can also be effectively accounted for. Numerically stable algorithms are used to determine the left nullspace basis of a rational transfer matrix and then synthesize the residual generators. For a TVC system, the consideration of the nozzle load together with the EMA dynamics imply the modeling of the structural compliance between the vehicle skirt and the EMA anchorage point, and the presence of the so-called vehicle-induced loads produced by the rocket motion itself.

It is essential to properly assess the designed filters in fault-free scenarios to not trigger false alarms. To verify the FDI performance, a (nonlinear) physical high-fidelity model has been implemented using the Modelica modeling language and integrated within Simulink<sup>®</sup>, here adopted as main simulation environment. Robustness against model uncertainties is proven with a small Monte-Carlo simulation campaign in a standalone context, *i.e.* without the full vehicle model. In the same framework, the detection triggering is verified when sensor faults are injected. The filter robustness is lastly assessed using the CALLISTO RLV simulator employing a physical model of the whole rocket (including the aforementioned TVC advanced model), such that the filter is stimulated with representative inputs.

The paper continues in Section 2 with a detailed formulation of the problem and the EMA-based TVC system modelling, including the mechanical load exerted by the rocket engine. Thereafter the mathematical model of an EMA-based TVC system is introduced in Section 3. The FDI synthesis framework is introduced and the application of the nullspace-based synthesis strategy is explained in Section 4, including considerations about isolability of the faults (Section 4.1). In Section 5, the residual generator performance is then assessed in both fault-free and faulty scenarios, and the proposed solution capabilities benchmarked at the example of an RLV mission scenario using a high-fidelity TVC physical model. In Section 6 conclusions are drawn.

# 2. Problem formulation

To properly modify the engine's thrust vector direction, the TVC actuation system must be able to tilt the engine's nozzle for the correct pointing as commanded by the GNC algorithms. For this reason, the engine is commonly mounted on a gimbal system and maneuvered by two linear actuators in a closed kinematic chain designed to best comply with the EMA load limits, as well as the required speed and maximum displacement operational domains. The actuator therefore must employ an embedded closed-loop control system to be able to track the deflection commands using an appropriate set of sensors. The EMA-based TVC system comprises the following hardware:

- 1. the engine nozzle, with its structural supports, anchorage points and mechanical joints;
- 2. a DC power source and the conditioning power electronics;



Figure 1: Representation of an EMA-based TVC system architecture.



Figure 2: Typical EMA stroke displacement control for TVC deflection command tracking.

- 3. the embedded actuator control computer (ACC), in charge of controlling the EMA stroke position;
- 4. two EMAs (one per axis) to physically move the engine nozzle.

At the same time, an EMA is made by different subsystems to produce the required energy transformations and convert the electrical power into mechanical force. Three building components are normally identified:

- the power drive electronics (PDE), which manages the power flow between the electrical supply and actuation system;
- the electric motor (EM), considered as a 3-phase Permanent Magnet Synchronous Motor (PMSM), to obtain a rotational motion from the provided electrical power;
- a mechanical power transmission (MPT) stage to convert the motor rotational motion into a translational motion;
- a set of sensors to be able to control the actuator position, composed by a current sensor, a motor rotational speed sensor, and a stroke displacement position (LVDT) sensor.

The described architecture is synthetically depicted in Figure 1. Note that there exist different MPT configurations: For instance, EMAs may employ a gearbox with appropriate transfer ratios to suitable modify the motor speed/torque domain. Here a direct-drive linear EMA is considered without any gearing element, because of the higher geometrical integration potential, reduced friction and backlash effects, and better efficiency, stiffness and reliability. This solution has also been adopted by the VEGA launch vehicle.<sup>13</sup>

The control system can be freely designed. As a reasonable compromise between simplicity and disturbance rejection properties, typically three nested loops made of PID controllers are employed. Here the following structure is assumed: an outer loop controls the EMA stroke position to obtain the desired nozzle deflection angle as commanded by the GNC system. It must rely on the measurements of a LVDT sensor to produce a suitable velocity reference signal. Another control loop deals with the velocity tracking, exploiting the measurements obtained by the EM sensor(s). Lastly, the innermost loop drives the EM torque control based on the sensed EM current. A depiction of the overall control scheme is shown in Figure 2.

# 3. Modeling of an Electro-Mechanical Actuator-based TVC system

In this Section an FDI synthesis model of the TVC system is derived. It is assumed that the TVC axes are independent, therefore cross-couplings between the two vectoring directions are not taken into account. The overall TVC architecture



Figure 3: TVC system architecture for linear modeling.

for the development of a linear synthesis model for fault detection is shown in Figure 3. A PDE model is not proposed, since the three-phase inverter switching dynamics lies in a much higher frequency range than the operative EMA one.

#### 3.1 Electric motor modeling

The electric motor balance of torques can be expressed as

$$J_r \ddot{\theta}_m = C_m = C_{em} - C_{m,\text{load}} - C_{\text{loss}},\tag{1}$$

where  $J_r$  is the rotor moment of inertia,  $\theta_m$  the rotor angular displacement,  $C_{em}$  is the electromagnetic torque produced by the motor itself,  $C_{m,load}$  the torque produced by the load connected to the motor, and finally  $C_{loss}$  is the torque loss due to the numerous dissipative or disturbance effects, including friction, magnetic losses, hysteresis effects and cogging torque.<sup>24</sup> The latter term is hereafter discarded for the purpose of building a linear model, but reintroduced afterwards in the nonlinear simulation model used for validation. The load on the motor depends on the counteracting force exerted onto the EMA screw; taking  $L_g$  as the mechanical (rotational to translational) transmission ratio and introducing a mechanical efficiency factor  $\eta_e$ , the motor load torque can be expressed as

$$C_{m,\text{load}} = L_g F_e / \eta_e,\tag{2}$$

where  $F_e$  is the load force acting on the screw, which is detailed in the next Section.

To determine the expression of  $C_{em}$ , the stator current equations must be written. For simplicity an isotropic machine is considered. It is known<sup>24, 25</sup> that for a PMSM, the three-phase stator and rotor quantities can be transformed into a single rotating reference frame to eliminate the effect of time-varying inductances. This procedure transforms a three-windings machine into a two-phase machine equipped with two windings, which is known as *direct-quadrature-zero* transformation<sup>2</sup>. As such:

$$v_d = R_m i_d + \dot{\Psi}_d - \dot{\theta}_m L_m i_q, \tag{3}$$

$$v_q = R_m i_q + \Psi_q + \theta_m L_m i_d + \theta_m \Psi_{pm}, \tag{4}$$

$$\Psi_d = L_m i_d + \hat{\Psi}_{pm},\tag{5}$$

$$\Psi_q = L_m i_q. \tag{6}$$

where the subscript 'd' and 'q' denote the direct and quadrature axes, respectively; the related currents, voltages and magnetic fluxes are expressed with v, i and  $\Psi$ , respectively;  $L_m$  and  $R_m$  are the winding inductance and resistance, respectively, while  $\hat{\Psi}_{pm}$  is a term capturing the magnetic flux related to the permanent magnets. By developing overall machine power equation,<sup>24</sup> one obtains a simplified electromagnetic torque expression as:

$$C_{em} = n_p \hat{\Psi}_{pm} i_a = k_t i_a,\tag{7}$$

<sup>&</sup>lt;sup>2</sup>Direct axis 'd' fixed with the North of the permanent magnets, and the quadrature 'q' is perpendicular to it.

where  $n_p$  is the number of the magnetic pole pairs and  $k_t$  is known as the motor torque constant. Equation (7) implies that  $C_{em}$  depends only on  $i_q$ , therefore the relevant current dynamics can be extracted by composing Equations (4) and (6) and by neglecting the cross-coupling terms as:

$$v_q \simeq R_m \dot{i}_q + L_m \dot{\dot{i}}_q \implies \dot{i}_q \simeq -\frac{1}{L_m} (R_m \dot{i}_q + v_q).$$
 (8)

The input voltage  $v_q$  is often regulated by a PI controller with output saturation to respect the physical voltage limits and anti wind-up action for disturbance rejection and to counteract the neglected coupling terms.

#### 3.2 Mechanical power transmission and load modeling

The MPT model simply considers the conversion of the motor torque, angular speed and angular displacement into a force, translational velocity and displacement, as:

$$y_a = L_g \theta_m, \tag{9}$$

$$v_a = L_g \dot{\theta}_m,\tag{10}$$

$$F_a = L_g^{-1} C_m - F_{\text{loss}},\tag{11}$$

where  $y_a$  and  $v_a$  are the stroke displacement (from its neutral position) and velocity,  $F_a$  the applied force to the load and  $C_m$  is from Equation (1). The term  $F_{loss}$  contains the dissipation effects and nonlinear characteristics that are in this context ignored. The housing, anchorage and joints stiffness imply a non-negligible dynamic effect which can be captured using a fictitious spring acting between the EMA screw and the load (nozzle). Therefore, this elastic force can be expressed as

$$F_e = k_s(y_a - y_e) \tag{12}$$

where  $k_s$  is an equivalent spring coefficient and  $y_e$  the translation of the nozzle which, for small deflection angles, can be expressed as

$$y_e = L_a \beta, \tag{13}$$

with  $\beta$  being the nozzle angular deflection for the considered EMA and  $L_a$  the actuator lever arm about the gimbaling point.

The nozzle dynamics depends on the displacement produced by the EMA dynamics, it's own inertial effects, but also the effect of another equivalent spring-damper mechanism at the gimbal point with constants  $k_e$  and  $b_e$ . This means that the dynamics about the gimbal point is

$$\alpha^{-1}\ddot{\beta} = F_e L_a - k_e \beta - b_e \dot{\beta},\tag{14}$$

where  $\alpha$  is a term defining the inverse of the nozzle inertia with respect to the gimbal. Hence

$$\alpha = \frac{1}{J_e + M_e L_e},\tag{15}$$

with  $J_e$  being the nozzle moment of inertia with respect to its center of mass (placed at distance  $L_e$  from the gimbal point) and  $M_e$  its mass.

#### 3.3 TVC linear model

The whole TVC system model can be finally obtained. Note that, after neglecting  $C_{loss}$  and  $F_{loss}$ , the equations are linear. The state-space representation can be derived: by considering Equation (8) to describe the current dynamics; by combining Equations (1), (2), (7), (12) and (13) to obtain the motor dynamics; from Equations (12) to (14) for the nozzle dynamics. The sign of  $\beta$  depends on the EMA placement; in this study, it is considered that a positive EMA stroke displacement induces a positive gimbal angle. The resulting state-space system is:

$$\dot{x}_{1}(t) = x_{2}(t)$$

$$\dot{x}_{2}(t) = \alpha \left[ -(k_{e} + L_{a}^{2}k_{s})x_{1}(t) - b_{e}x_{2}(t) + k_{s}L_{g}L_{a}x_{3}(t) \right]$$

$$\dot{x}_{3}(t) = x_{4}(t)$$

$$\dot{x}_{4}(t) = J_{a}^{-1} \left[ L_{g}k_{s}L_{a}\eta_{e}^{-1}x_{1}(t) - L_{g}^{2}k_{s}\eta_{e}^{-1}x_{3}(t) - b_{a}x_{4}(t) + k_{t}x_{5}(t) + d(t) \right]$$

$$\dot{x}_{5}(t) = L_{m}^{-1} \left[ -R_{m}x_{5}(t) + u(t) \right]$$
(16)



Figure 4: Diagram of the adopted residual generation approach.

$$y_{1}(t) = x_{3}(t)$$

$$y_{2}(t) = x_{4}(t)$$

$$y_{3}(t) = L_{g}x_{3}(t)$$

$$y_{4}(t) = x_{5}(t).$$
(17)

The state vector is therefore  $x = [x_1, ..., x_5]^T = [\beta, \dot{\beta}, \theta_m, \dot{\theta}_m, i_q]^T$ . Note that all the moments of inertia are expressed with respect to the gimbal point S. The outputs depend on the available sensors, which are the motor angular displacement, the motor angular rate, the stroke displacement and the motor current. Therefore the output vector is  $y = [y_1, ..., y_4]^T = [\theta_m, \dot{\theta}_m, y_a, i_q]^T$ . The inputs are the motor quadrature voltage  $(u = v_q)$  and an additive term including the disturbance d(t) acting on the TVC. The latter correspond to the vehicle-induced loads, hence the reaction forces and torques applied at the gimbal point as consequence of the rocket motion. The linear model can be rewritten in the classic matrix form:

$$\dot{x}(t) = Ax(t) + b_u u(t) + b_d d(t)$$

$$y(t) = Cx(t)$$
(18)

*Remark* 1. Further disturbances to the TVC dynamics are, in this work, not considered. They may include, for instance, aerodynamic effects acting on the nozzle. Their magnitude is here considered negligible with respect to the vehicle-induced loads.

## 3.4 Residual generator synthesis model

The fault detection process focuses on the sensors whose degradation can affect the closed-loop performance. Therefore, the motor angle  $\theta_m$  is used as available measurement from the residual generator, but only motor velocity, motor current and actuator position are monitored for additive faults. As such, the outputs of Equation (18), must be augmented with the fault inputs:

$$\dot{x}(t) = Ax(t) + b_u u(t) + b_d d(t)$$

$$y(t) = Cx(t) + f(t).$$
(19)

where the vector  $f(t) = [0, f_1(t), f_2(t), f_3(t)]^T$  is consists of generic input signals; this implies that the synthesis is not limited to specific fault types.

# 4. FDI residual generator synthesis

In the state space expressed in Equation (19), the load d(t) acting on the nozzle, and thus indirectly affecting the motor dynamics, is considered as unknown disturbance that has to be perfectly decoupled from the outputted residual signals. To simplify the synthesis procedure description, the augmented state-space can be rewritten in the Laplace domain in input-output form as

$$\mathbf{y}(s) = G_u(s)\mathbf{u}(s) + G_d(s)\mathbf{d}(s) + G_f(s)\mathbf{f}(s),$$
(20)

where the quantities  $\mathbf{y}, \mathbf{u}, \mathbf{d}, \mathbf{f}$  have the same meaning as before, but written in the Laplace domain, whereas  $G_u, G_d$  and  $G_f$  the derived transfer function matrices. In the case under study,  $G_f$  has dimensionality  $4 \times 3$  and has no dynamics (hence constant), since the faults directly affect the output. On the other hand,  $G_u$  and  $G_d$  have dimension  $4 \times 1$ . The fault detection and isolation problem requires a residual generator which can be written as

$$\mathbf{r}(s) = Q(s) \begin{bmatrix} \mathbf{y}(s) \\ \mathbf{u}(s) \end{bmatrix},\tag{21}$$

where  $\mathbf{r}(s)$  are the residual signals (in Laplace domain) and Q(s) the filter to be synthesized, which must be stable (only poles with negative real part) and proper. The fault detection problem is addressed by producing residuals

$$r(t) = \begin{bmatrix} r_1(t) \\ r_2(t) \\ r_3(t) \end{bmatrix}$$
(22)

different from zero whenever a fault occurs. The residuals are then compared with a threshold to determine if the amplitude of the residual is "large enough" to avoid false alarms. When the residual vector is sensitive to all faults, the *complete fault detectability* property is achieved. Furthermore, when faults enter in the system dynamics as signals with a clear frequency signature, *i.e.* there exist a set of frequencies (including zero) that defines the fault signals (for example to detect bias or oscillatory faults), then the *strong fault detectability* property is desirable. These properties are formalized in the next Section. On the other hand, isolability concerns the location of the fault, hence the determination of the presence of a specific fault by associating the raise (or non raise) of a specific residual signal  $r_j$  (or more residuals) with a specific fault  $f_j$ . Normally, it is desirable that multiple occurring faults can be detected at the same time without altering the isolation capability: when this happens, the property is be called *strong fault isolation*.

*Remark* 2. Another problem formulation could include the vehicle-induced load disturbances d(t) as an additional fictitious control input. This is considered an option because the loads can be computed knowing the vehicle states. However, the load computation depends on both TVC deflection values, but the latter are normally determined directly from  $y_a$  or  $\theta_m$  measurement. In presence of faults on one of these two quantities (especially on  $\theta_m$  that is not monitored), also the load computation would lead to wrong results, thus jeopardizing the FDI system.

#### 4.1 Detectability and isolability analysis

We desire *strong complete fault detectability* and *strong fault isolability* properties. It is convenient to rewrite Equation (21) after substituting Equation (20):

$$\mathbf{r}(s) = Q(s) \begin{bmatrix} G_u(s) & G_d(s) & G_f(s) \\ 1 & 0 & 0 \end{bmatrix} \begin{vmatrix} \mathbf{u}(s) \\ \mathbf{d}(s) \\ \mathbf{f}(s) \end{vmatrix}.$$
 (23)

Complete fault detectability can be achieved<sup>22</sup> if and only if:

$$\operatorname{rank} \begin{bmatrix} G_d(s) & G_{f_j}(s) \end{bmatrix} > \operatorname{rank} (G_d(s))$$
(24)

with  $j \in [1, 2, 3]$  representing the *j*-th column of the related matrix. This is generally fulfilled whenever the output measurements are more than the disturbance inputs, which is the case of the system under study. For strong fault detectability a set of complex frequencies of interest  $\Omega$  must be defined. Only the set  $\Omega = \{0\}$  is considered, thus detectability of constant faults (*i.e.* biases) is checked. Let us define:

$$G_{e_j}(s) := \begin{bmatrix} G_{f_j}(s) & G_u(s) & G_d(s) \\ 0 & 1 & 0 \end{bmatrix}$$
(25)

and

$$\bar{G}_{e_j}(s) = \begin{bmatrix} G_{e_j}(s) \\ I_{1\times 3} \end{bmatrix},\tag{26}$$

with  $I_{1\times 3}$  being a truncated identity matrix [1,0,0]. Strong fault detectability is achieved when every fault  $f_j$  is detactable and when the matrix  $G_{e_j}$  has the same zero structure of  $\overline{G}_{e_j}$  for every  $s \in \Omega$ . This latter condition is also verified for the system in (19).

Massurement and fault configurations			Detectability		Isolability		
Measurement and fault configurations		Complete	Strong	No	Weak	Strong	
#1	Outputs:	$[\theta_m, \dot{\theta}_m, y_a, i_q]$	$\checkmark$	$\checkmark$			./
	Faults:	$[\dot{ heta}_m, y_a, i_q]$					v 
#2	Outputs:	$[\dot{ heta}_m, y_a, i_q]$	$\checkmark$	$\checkmark$	$\checkmark$		
	Faults:	$[\dot{ heta}_m, y_a, i_q]$					
#3	Outputs:	$[\dot{\theta}_m, y_a, i_q, F_e]$	√	$\checkmark$			
	Faults:	$[\dot{ heta}_m, y_a, i_q]$					V
#4	Outputs:	$\begin{bmatrix} \dot{\theta}_m, y_a, \dot{i}_q, F_e \end{bmatrix}$	 ✓	$\checkmark$		$\checkmark$	
	Faults:	$[\dot{\theta}_m, y_a, i_q, F_e]$					
#5	Outputs:	$\begin{bmatrix} \theta_m, \dot{\theta}_m, y_a, i_q, F_e \end{bmatrix}$	 √				✓
	Faults:	$[\dot{\theta}_m, y_a, i_q, F_e]$					

Table 1: Detectability and isolability of faults within a EMA-based TVC system based on different measurements and fault configurations.

For strong fault isolability one can define the fault signature matrix *S* where each element (*k*-th column, *i*-th row) is populated as follows:

$$S_{k,i} = \begin{cases} 1 & \text{if } R_{f_i}^J(0) \neq 0\\ 0 & \text{if } R_{f_i}^j(s) = 0\\ -1 & \text{if } R_{f_i}^J(0) = 0 \land R_{f_i}^j(s) \neq 0, \end{cases}$$
(27)

where  $R_{f_i}^j$  is the transfer function from *i*-th fault to *j*-th residual. When an entry is '1', it means that when a fault occurs, the steady value of the residual output is nonzero, otherwise the residual displays the fault only "temporarily" and then the entry is '-1'. In case the residual is not affected by the fault at all, then '0' is set. Clearly the ideal case is when  $S = I_{3\times 3}$ , which occurs only when:<sup>22</sup>

$$\operatorname{rank} \begin{bmatrix} G_d(s) & G_f(s) \end{bmatrix} = \operatorname{rank}(G_d(s)) + m_f, \tag{28}$$

with  $m_f$  being the number of outputs. For the system in (19), Equation (28) can be trivially verified.

However, it is natural to wonder if this is the only possible measurements-faults combination. For example, if  $\theta_m$  output was also monitored for faults, could the strong fault detectability and isolability properties be kept? This is also of great relevance if  $\dot{\theta}_m$  is obtained by differentiating  $\theta_m$ . In this case, it may be possible that a fault affecting  $\theta_m$  also affects  $\dot{\theta}_m$  signal. Therefore, it is advantageous to define which combinations of measurements and input faults are feasible for *strong* FDI. This analysis is summarized in Table 1. Case #1 represents the situation so far investigated; Case #2 is concerned with the eventuality that  $\theta_m$  measurements are removed from the problem formulation. In this case, it is not possible to achieve fault isolation at all. The only residual completely unaffected by this change is the motor current; the other residuals are coupled. As such, there exist no such residual that does not get triggered by a fault on  $\dot{\theta}_m$  and not on  $y_a$ , and vice-versa. Case #3 explores a new possibility: it is not rare that a TVC system includes a force sensor between the EMA and the nozzle, *i.e.* at the anchorage point. This is described by Renault<sup>26</sup> and can be effectively employed to augment the EMA control system to dampen the nozzle oscillations. When this quantity is available, it can be added in the outputs in Equation (17) as

$$y_5 = F_e = -L_a k_s x_1 + L_g k_s x_3 \tag{29}$$

and consequently included in the FDI problem formulation. This restores the strong fault isolation property of Case #2. If also faults on the force sensor are to be determined, the latter property is lost again. In this case weak fault isolability can be achieved, up to one fault can be isolated (Case #4). If from Case #4 we relax the  $\theta_m$  output elimination assumption, an arbitrary number of faults can be detected and isolated again, since another measurement is available (Case #5).

*Remark* 3. The presence of the disturbance input d makes a great difference in the fault detectability and isolability properties. Excluding it from the problem formulation could be beneficial only when its amplitude is relatively small with respect to the impact it has on the overall dynamics. In the TVC case, the vehicle-induced loads are definitely not negligible. This is also why, during the control system design of a rocket, the so-called tail-wag-dog effect is especially

taken into account to counteract the dynamics from the nozzle swinging. This is indeed the dual effect of the described TVC vehicle-induced loads.

#### 4.2 The nullspace method

The nullspace method is considered in this work as approach to derive a suitable dynamic filter for the computation of the residual signals. This is extensively described by Varga.<sup>22</sup> Solving the the Exact Fault Detection and Isolation Problem (EFDIP) is hereafter considered. It allows the decoupling of both disturbances and inputs, and provides filters of minimal order, a desirable property for the final implementation into the hardware and for disturbance rejection. It is composed by three steps, at the end of which the filter Q(s) gets updated as  $Q(s) \leftarrow Q_K(s)...Q_1(s)$  with K being the step index.

The first step is to determine a basic fault detection filter. This basic filter can be seen as a composition of three different dynamical systems, each dealing with one fault as

$$Q(s) = [Q^{1}(s), Q^{2}(s), Q^{3}(s)]^{T}.$$
(30)

For each sub-filter to achieve perfect decoupling of inputs, disturbance and all the "non-related" faults, the following condition obtained from Equation (23) must be fulfilled:

$$Q^{i}(s) \begin{bmatrix} G_{u}(s) & G_{d}(s) & \tilde{G}_{f,i}(s) \\ 1 & 0 & 0 \end{bmatrix} := Q^{i}(s)G_{e}(s) = 0,$$
(31)

where  $\tilde{G}_{f,i}(s)$  is a fault transfer matrix where all faults with index different than *i* have been excluded. Equation (31) means imposing that  $Q^i(s)$  must be a left-annihilator of  $G_e(s)$ . This implies that it can be solved by determining a left nullspace basis of  $G_e(s)$  which is able to fulfill the decoupling conditions while allowing the *i*-th filter to be affected by the *i*-th fault.

The second step is meant to reduce the order of Q(s) by selecting a suitable transfer function matrix  $Q_2(s)$  that makes  $Q_2(s)Q(s)$  have the least (McMillan) degree. The idea is to try out different polynomial basis vector combinations (with different degrees) iteratively, to make sure that the fault becomes visible with the least order one.

The last step is to eventually stabilize the filter and impose the desired dynamics. Therefore, a proper and invertible  $Q_3(s)$  is designed using a left coprime factorization, such that the resulting final filter  $Q_3(s)Q(s)$  has the desired dynamics. In this case, it has been imposed that the poles of the filter are set to -10, as a good compromise between detection speed and filtering eventual high frequency measurement noise.

#### 4.3 Analysis of the synthesized residual generators

The filter  $Q^1(s)$  is in charge of detecting faults on the motor angular velocity. It is a first order transfer function that compares the filtered and differentiated motor position with the measured angular velocity. This means that the filter has a zero in s = 0. On the other hand,  $Q^2(s)$  is a constant filter directly comparing the motor position with the stroke position multiplied by a gain. The additional plant output  $\theta_m$  (not subject to faults) is basically acting as hardware redundancy to directly check  $y_a$ . Lastly,  $Q^3(s)$  is again a first order filter; it must be noticed that in this case the pole could not be imposed, because the residual is obtained by subtracting the dynamic effect of a voltage input in terms of predicted current, to the current measurement itself. In conclusion, the dynamics is dictated by the current one, as per Equation (8). Therefore, the overall filter Q(s) is second order when assembling all the filters together. For the other cases considered in Section 4.1, namely #3 and #6, the structure of the filter becomes slightly different, but the order of each individual sub-filters never overcomes three. Hereafter, only Case #1 is considered.

## 5. Simulation results

In this Section the performance of the FDI filter is assessed. The simulation setup includes a nonlinear high-fidelity TVC model developed using Modelica, an open-souce multi-physics acausal modeling language. This has several advantages: *a*) it enables a clear representation of physical meaning within an object-oriented structure; *b*) a free Modelica Standard Library (MSL) exists<sup>27</sup> for the creation of multibody models, entailing several benefits for the TVC modeling; *c*) advanced friction models together with backlash effects can be obtained.<sup>25</sup> The TVC model employed here is included in DLR's VLVLib.<sup>28,29</sup> In the Modelica model, the following effects are included (grouped within  $F_{\text{loss}}$  in Equation (11)):





Figure 5: Reference signals for the fault-free Monte-Carlo campaign; motor load and voltage signals refer to the unperturbed case.

- a load-dependent friction model: it includes several nonlinear effects, like the Coulomb and Stribeck friction, as well as a viscous (linearly velocity-dependent) component. In addition, it is known<sup>30,31</sup> that the load alters the friction properties. In this work, this aspect has been considered as well, *cf.* Marè *et al.*<sup>30</sup> and Farì *et al.*<sup>25</sup>
- backlash effect: it is known that EMAs are subject to backlash, hence a "loss of motion" happening while the mechanical components are not in contact. Normally, in modern EMAs, this effect is generally small, but it has the potential of degrading the FDI performance;
- the EMA screw mass, that adds an additional small load onto the motor.

At the same time, the EM model includes the following effects (grouped within  $C_{loss}$  in Equation (1)):

- a friction model, that includes the Coulomb and Stribeck friction, as well as a viscous (linearly velocity-dependent) component;
- the hysteresis losses and cogging torque effect.

This nonlinear model is included in all the following setups: at first, the FDI robustness is assessed in a fault-free scenario. Then, it is shown that the filter reacts and correctly responds to the injected faults. Lastly, the full end-to-end mission of CALLISTO VTVL RLV<sup>32</sup> is used to prove that, in a fault-free situation, the FDI filter has a good performance when solicited by representative inputs and disturbance.



Figure 6: Residual signals in a fault-free Monte-Carlo simulation campaign using a high-fidelity TVC model.

## 5.1 Fault-free scenario

At first, the case where no faults are present is discussed, as it is essential to prove that the performance of the FDI filter are robust in the nominal case (with vehicle-induced disturbances), but also against the aforementioned unmodeled effects and parametric uncertainties. The TVC model with the EMA nonlinear dynamics is compiled as a Functional Mock-up Unit (FMU) and integrated within a Simulink<sup>®</sup> simulator. A Monte-Carlo campaign of 100 runs is then performed; all parameters have been perturbed with a uniform distribution in the range of 10%-20% of their nominal value. The injected TVC deflection angle reference is reported in Figure 5, together with the reference stroke displacement command, the motor loads and the commanded voltage in the nominal case. Note that the plots have been normalized. The vehicle-induced loads acting on the TVC are extracted from a high-fidelity simulation of the launch phase and injected in the simulation. They act exclusively when the  $\beta$  deflection angle is different than zero; this is a normal consequence of the fact that the action-reaction forces between the vehicle main body and the nozzle due to the vehicle acceleration are present only if the center of mass of the nozzle is not longitudinally aligned with the one of the main body.

The FDI performance depends upon the choice of suitable thresholds for the residuals. They are not uniquely defined since they depend on the amplitude of faults which are considered relevant. The thresholds are selected to correspond to 3-10% of the normal operation range of the motor velocity, stroke displacement and motor current.

Figure 6 shows the three resulting residual signals. As expected they are not zero as applied to a nonlinear model with parameter perturbations. Despite that, they remain lower than the selected thresholds. It can be seen how the current signal is the one more close to its thresholds and therefore has to be treated with care, for example when trying to estimate the fault amplitude.





(a) Monte-Carlo campaign with a bias fault on  $\dot{\theta}_m$  sensor.

(b) Monte-Carlo campaign with a bias fault on  $y_a$  sensor.





Figure 7: Residual signals in presence of a bias fault using a high-fidelity TVC model.

## 5.2 Fault simulations

For faults injection, the same simulation setup is used. Three different Monte-Carlo simulation campaigns (100 runs) have been ran, each with a bias fault introduced at about one third of the overall simulation time into  $\dot{\theta}_m$ ,  $y_a$  and  $i_q$  measurements, respectively. The results are depicted in Figure 7. It can be noticed that  $r_1$  residual behaves as a first order system, in accordance to what expected. The stroke displacement residual  $r_2$  instead has no dynamics. Lastly, the motor current residual  $r_3$  has the form of a step. For the reasons explained in Section 4.3 that the voltage dynamics is fast, the result is a filter with a very small time constant. In all cases the selected threshold is exceeded by at least 20%, which is an acceptable robustness margin with respect to the chosen thresholds. Note that for the achieved strong isolation property, the residual magnitude would have not differed substantially if the fault was injected in the same simulation.

The most important consequence for faults affecting EMA sensors is that closed-loop performance can be heavily degraded. Figure 8 shows the case of an EMA step position command (implying a step in  $y_a$ ) at about one fourth of the pseudo-time and a bias fault affecting  $y_a$  sensor at about half the simulation time. It can be seen that the EMA controller immediately reacts to the fault by adjusting the voltage output and compensate for the bias. However, this means that the actual position of the stroke is different than the one supposed by the flight control system. In turn, depending on the TVC lever arm  $L_a$ , this can result in a very degraded tracking of the  $\beta^*$  TVC deflection commands for vehicle control.



Figure 8: Consequence at closed-loop TVC control system level after a fault affecting  $y_a$ .

## 5.3 Fault-free scenario in an end-to-end RLV mission simulation

To confirm that the FDI filter can be applied to the scenario of a reusable launch vehicle, the full end-to-end high-fidelity CALLISTO simulator is employed and ran contextually with the residual generator. Therefore, the filter is engaged with the inputs and disturbance of the CALLISTO vehicle, which undergoes the flight phases ascent, boostback, aero-dynamic re-entry and the engine re-ignition for the final landing and touchdown. The simulation is conducted with both vehicle and TVC nominal parameters. The advanced model adopted previously is here included for both TVC directions. The vehicle is modeled using Modelica as well: this includes the vehicle structural dry mass and several masses representing the propulsion fluidics masses (with time-varying position as the fuel is depleted). This makes it a multibody model; note that, with such a modeling, the two EMAs may slightly disturb each other during simultaneous thrust vectoring. Figure 9 shows the final results for one of the two TVC actuators (the other would exhibit as very similar). The good residual performance is confirmed for a fault-free case, and it is shown that the filter is almost insensitive to the vehicle dynamics and that local monitoring is not affected by the vehicle-induced loads which have been sufficiently decoupled. A Monte-Carlo analysis is here not performed because the simulation computational burden is very high, thus requires a considerable amount of time. This aspect will be taken into account in future works.

## 6. Conclusions

This paper presented an effective methodology for the synthesis of a fault detection and isolation filter targeting the sensors included in the electro-mechanical actuators maneuvering a TVC system. Assessing performance degradation to the TVC closed-loop control system is essential to secure reliable performance and have successful missions. Exploiting this additional information can lead to in-flight control laws adjustment to accommodate the fault, or post-flight analysis, inspection and maintenance to keep the actuator fully operative.

An analysis of fault detectability and isolability of the faults affecting the motor angular velocity sensor, the stroke displacement sensor and the motor current sensor has been performed. It has been shown that a measurement of the motor angle (*e.g.* via a angular resolver) enables *complete strong fault detectability* and *strong fault isolability* of the sensors, whose measurements are employed in the EMA control system. Scenarios with different measurements/fault-requirements combinations have been discussed. The nullspace method for filter synthesis has proven effective for successfully decoupling the plant input, measurement and disturbance (vehicle-induced loads) from the residual signal. The performance of the filter has been tested in fault-free scenarios with a Monte-Carlo campaign and robustness has been successfully proved in all cases in both a standalone simulator and applied to a full end-to-end reusable launch vehicle mission. It has been shown how the residual correctly identifies the fault occurrence when sensor biases are artificially injected.

## Acknowledgments

This research is funded by the European Space Agency (ESA) – Ph.D. grant nr. 4000134582/21/NL/GLC – and the German Aerospace Center (DLR).





Figure 9: Fault-free residual signals in an end-to-end high-fidelity simulation of a VTVL RLV mission.

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